# micromachines 

ISSN 2072-666X
www.mdpi.com/journal/micromachines

## Article

# High-Performance Shuffle Motor Fabricated by Vertical Trench Isolation Technology 

Edin Sarajlic ${ }^{1, *}$, Christophe Yamahata ${ }^{2}$, Erwin Berenschot ${ }^{1}$, Niels Tas ${ }^{1}$, Hiroyuki Fujita ${ }^{2}$ and Gijs Krijnen ${ }^{1}$

1 Transducers Science and Technology (TST) Group, MESA+ Research Institute, University of Twente, PO Box 217, 7500 AE Enschede, The Netherlands;
E-Mails: j.w.berenschot@ewi.utwente.nl (E.B.); n.r.tas@ewi.utwente.nl (N.T.);
g.j.m.krijnen@ewi.utwente.nl (G.K.)

2 Center for International Research on MicroMechatronics (CIRMM), Institute of Industrial Science, The University of Tokyo, 4-6-1 Komaba, Meguro-ku, Tokyo, Japan;
E-Mails: christophe.yamahata@epfl.ch (C.Y.); fujita@iis.u-tokyo.ac.jp (H.F.)

* Author to whom correspondence should be addressed; E-Mail: e.sarajlic @ewi.utwente.nl; Tel.: +31-53-7112700; Fax: +31-53-7112701.

Received: 19 May 2010; in revised form: 6 July 2010 / Accepted: 12 July 2010 /
Published: 16 July 2010


#### Abstract

Shuffle motors are electrostatic stepper micromotors that employ a built-in mechanical leverage to produce large output forces as well as high resolution displacements. These motors can generally move only over predefined paths that served as driving electrodes. Here, we present the design, modeling and experimental characterization of a novel shuffle motor that moves over an unpatterned, electrically grounded surface. By combining the novel design with an innovative micromachining method based on vertical trench isolation, we have greatly simplified the fabrication of the shuffle motors and significantly improved their overall performance characteristics and reliability. Depending on the propulsion voltage, our motor with external dimensions of $290 \mu \mathrm{~m} \times 410 \mu \mathrm{~m}$ displays two distinct operational modes with adjustable step sizes varying respectively from 0.6 to 7 nm and from 49 to 62 nm . The prototype was driven up to a cycling frequency of 80 kHz , showing nearly linear dependence of its velocity with frequency and a maximum velocity of $3.6 \mathrm{~mm} / \mathrm{s}$. For driving voltages of 55 V , the device had a maximum travel range of $\pm 70 \mu \mathrm{~m}$ and exhibited an output force of 1.7 mN , resulting in the highest force and power densities reported so far for an electrostatic micromotor. After five days of operation, it had traveled a


cumulative distance of more than 1.5 km in 34 billion steps without noticeable deterioration in performance.

Keywords: electrostatic micromotor; stepper micromotor; shuffle motor; inchworm motion; surface micromachining; trench isolation

## 1. Introduction

Microactuators are small-scale transducers that transform non-mechanical energy into mechanical work. A literature survey shows that, at the micrometer scale, electrostatic actuation is the most commonly used driving principle for these actuators [1,2]. This is primarily due to the favorable scaling properties of electrostatic force that allow for the generation of relatively large energy densities [3]. In addition, electrostatic microactuators are essentially voltage driven. Consequently, fast electronic switching combined with small sizes results in fast responses and low power consumption. Lastly, one of the major reasons behind the widespread use of electrostatic microactuators is their compatibility with existing micromachining methods.

Electrostatic microactuators with relatively large output forces can be developed by exploiting the large electric fields achievable in narrow gaps [4]. However, the achievable electric field drops off rapidly with an increase in spacing between electrodes. As a result, electrostatic microactuators with large displacement ranges generally suffer from low output forces. Electrostatic micromotors based on stepping motion [5-13] are a practical solution for generating large output forces and large displacements simultaneously. These motors use a "large force-short stroke" microactuator to produce small, powerful steps. A clamping mechanism, either based on frictional force [5-8,10-13] or obtained with gear teeth [9], is employed to create large displacements via incremental steps. In these motors, the propulsion and the clamping are dissociated to allow for individual optimization of these two mechanisms. These motors have inherently more accurate positioning and higher reliability than scratch drives [14-16] and impact micromotors [17]. They also produce larger output forces than frictionless 3-phase stepper motors [18,19].

To enable a further increase in output force and positioning resolution while preserving large displacement ranges, electrostatic linear stepper micromotors with a built-in mechanical leverage have been developed. Shuffle motors [8,10], contraction beams motors [12] or stepper motors with an in-plane angular inflexion conversion [13] are a few examples of such mechanisms. Shuffle motors employ electrostatic force to inflect an elastic plate and generate small, powerful steps. A voltage controlled clamping mechanism is employed to accumulate steps in sequence, enabling bidirectional motion. The first shuffle motor was fabricated by a surface micromachining process using seven lithographic masks and three polysilicon device layers [8]. The motor, measuring $400 \mu \mathrm{~m} \times 560 \mu \mathrm{~m}$, was driven successfully up to a speed of $0.1 \mathrm{~mm} / \mathrm{s}$, corresponding with a cycling frequency of 1.16 kHz and an average step size of 85 nm . An output force of $43 \mu \mathrm{~N}$ was achieved at driving voltages of 25 V and 40 V for the plate and clamps, respectively. The performance and reliability of the motor were limited by charge accumulation in the silicon nitride layer employed as insulation between the
electrically grounded shuttle and the driving electrodes. The negative effect of charge accumulation could be ameliorated in an improved shuffle motor design [10]. In the design, charge accumulation was avoided by allowing mechanical contacts only between electrically grounded parts. The motor, measuring $200 \mu \mathrm{~m} \times 1500 \mu \mathrm{~m}$, could achieve a force of 0.45 mN at driving voltages of 65 V and 150 V for the plate and clamps, respectively. A large stroke of $\pm 100 \mu \mathrm{~m}$ was achieved. The motor was fabricated by the five-level SUMMiT V process ( 14 masks and five polysilicon layers), which is the most complex polysilicon surface micromachining technology reported to date [20].

In this work, we present a new design for the shuffle motor. The shuffle motors reported in the past $[8,10]$ were electrically grounded and they moved over predefined paths that served as individual counter electrodes for the plate and for the clamps. In our design, these underlying paths have been eliminated. Our motor moves over an unpatterned, electrically grounded surface, while driving voltage signals are directly applied to the plate and to the clamps. By combining this novel design with an innovative micromachining method based on vertical trench isolation [21], we have greatly simplified the fabrication of the shuffle motors. Furthermore, we have significantly improved their overall performance characteristics and reliability. Our motor has demonstrated exceptional performance in terms of force, resolution, displacement range and speed, leading to the highest force and power densities ever reported for an electrostatic stepper micromotor, as shown in Figure 1 (also see Table A1). In the following sections, we present the design and fabrication of the shuffle motor and derive analytical models to predict its performance characteristics. Finally, we demonstrate stepping motion of the motor and characterize its performance in terms of speed, average step size and output force.

Figure 1. Power-force performance of our shuffle motor (A) compared with previously published electrostatic micromotors (B-I). A: This work; B: Li et al. (2002) [16]; C: Sarajlic et al. (2006) [12]; D: de Boer et al. (2004) [10]; E: Yeh et al. (2002) [9]; F: Akiyama et al. (1997) [15]; G: Erismis et al. (2008) [13]; H: Tas et al. (1998) [8]; I: Kim et al. (2005) [11].



A This work
B L.Li (2002)
C E.Sarajic (2006)
D M.P.de Boer (2004)
E R.Yeh (2002)
F T.Akiyama (1997)
G M.A.Erismis (2008)
H N.Tas (1998)
I S.-H. Kim (2005)

## 2. Motor Design

The shuffle motor consists of two clamps that are mechanically connected by an elastic plate, as schematically illustrated in Figure 2.

Figure 2. Inchworm-like walking sequence of the shuffle motor. In the "contraction" phase, the voltage sequence is as follows: (i) Only the left clamp is activated; (ii) then, a voltage applied to the plate causes its inflexion, which generates powerful longitudinal contraction; (iii) the right clamp is then activated. In the "stretching" phase, the voltage sequence is as follows: (iv) the left clamp is released; ( $v$ ) the plate voltage is turned off, which causes stretching of the plate and motion of the left clamp; ( $v i$ ) both clamps are activated. For further motion to the left, the sequence is repeated from step $i$. Conversely, to obtain motion to the right, the sequence is reversed.


The plate and clamps are electrically insulated from each other, enabling individual biasing of these components. The motor moves over an unpatterned substrate covered with an insulating layer that is electrically grounded during operation. The driving voltage signals applied to the clamps and to the plate
are provided through the suspension flexures of the motor, which are not shown in Figure 2. As described above, the elimination of the underlying electrodes is a major improvement over previously reported shuffle motors [8,10]. Consequently, this reduces the complexity of the fabrication process (see Section 3 on Microfabrication). Furthermore, it allows for motion of the motor that is not confined to predefined paths. This was exploited in a shuffle motor with two degrees of freedom, as presented in [22].

A complete walking sequence of the shuffle motor is illustrated in Figure 2. An inchworm-like displacement is achieved by applying an appropriate voltage sequence to the clamps and plate. First, a voltage is applied between the left clamp and the grounded substrate, creating an electrostatic force in the direction normal to the substrate. The normal force induces friction in the activated clamp that prevents it from sliding (stage $i$ in Figure 2). Next, a voltage is applied on the plate, causing normal inflexion of the plate. The plate brings the right clamp closer to the left one, producing a "contraction" of the motor (ii). Since the normal inflexion of the plate is much larger than the longitudinal motion, there is a built-in mechanical leverage, resulting in a small but powerful contraction. Subsequently, the right clamp is activated to maintain its new position (iii) and the left clamp is released (iv). The plate is released and, by "stretching", it pushes the left clamp outwards (v). The actuation sequence is completed by the activation of the left clamp (vi). The entire sequence corresponds to the leftward motion of the whole motor in a single step. By repeating the actuation sequence continuously, a large number of steps can be accumulated, resulting in a large displacement range. Bidirectional motion of the motor is simply obtained by reversing the sequence.

## 3. Microfabrication

A successful operation of the proposed shuffle motor requires proper electrical insulation despite the mechanical connections between the clamps and the plate. To satisfy these requirements within a minimum number of processing steps, we have combined vertical trench isolation technology with conventional surface micromachining [21]. In this way, we propose a relatively simple fabrication process with only two low-stress polysilicon layers and four photolithography masks. This is a significant reduction in the process complexity compared with the fabrication processes reported by Tas et al. [8] who employed three polysilicon layers and seven masks, and de Boer et al. [10], who employed five polysilicon layers and 14 masks. The cross-section of the shuffle motor after release is depicted in Figure 3. For a detailed description of all processing steps, the reader is referred to [22].

Figure 3. A schematic view of the cross-section of the shuffle motor. The fabrication process combines vertical trench isolation technology and surface micromachining.


The motor employs trenches refilled with silicon nitride to provide an electrical insulation between the clamps and the plate, enabling the independent electrical biasing of these components. At the same time, the stiffness and mechanical strength of the electrical isolation trenches ensure mechanical integrity of the device. Silicon nitride isolation bumps are evenly spaced on the backsides of the clamps and plate. These bumps significantly reduce the contact area between the device and the silicon nitride coating on the wafer surface, preventing stiction of the clamps and plate during fabrication and operation. Furthermore, the reduced contact area also reduces contact charging and charge accumulation during operation. Finally, due to the hardness of silicon nitride, its use on both sides of the contact reduces wear, increasing reliability and durability of the motor.

Figure 4(a) is a SEM micrograph of a fabricated shuffle motor consisting of two U-shaped clamps bridged by an elastic rectangular plate. The silicon substrate underneath the motor was coated with a 210 nm thick silicon nitride layer. The motor is suspended by four flexures (two flexures per clamp), each of which plays both a mechanical and an electrical role. Mechanically, the beams are used as flexures to ensure the linear guidance of the motor. Electrically, they serve as conductive wires transmitting the driving voltage signals to the clamps and to the plate.

Figure 4. Micrographs of the shuffle motor. (a-c) are SEM pictures showing: (a) The shuffle motor with all components; (b) a cross-sectional view of a shuffle motor showing an isolating trench; (c) the backside of the plate with isolation bumps; (d) and (e) show operation of the motor. The repeating patterns integrated on the motor and the anchors are used for displacement measurement by Fast Fourier Transform (FFT) calculations (see details in Section 5).


Figure 4(b) is a cross-sectional view of the motor where we can distinguish two low-stress polysilicon layers: a thin layer of $1.15 \mu \mathrm{~m}$ for the plate and a thick layer of $5.50 \mu \mathrm{~m}$ for the clamps and the flexures. The thin layer results in a rather compliant plate required for proper operation of the motor. On the other hand, the thick layer ensures the rigidity of the clamps and a large out-of-plane stiffness of the flexure beams. In our design, the flexures are not electrically shielded from the grounded substrate. Therefore, the large out-of-plane stiffness is desirable to prevent pull-down of flexures during operation of the motor. In the same image, one can see that the plate is electrically insulated from the clamps by $2 \mu \mathrm{~m}$ wide isolation trenches refilled with silicon nitride.

Figure 4(c) shows a close-up view of the isolation bumps located on the backside of the plate. Due to the rounded shape of the rim of the bumps, the contact area is significantly reduced during operation of the motor. The isolation bumps protrude from the clamps and from the plate by a height of 210 nm . This height determines the distance between the activated clamp and the coated surface and limits the maximum inflexion of the plate. A maximum inflexion of $1.55 \mu \mathrm{~m}$ was measured for an initial air gap of $1.76 \mu \mathrm{~m}$ between the plate and the substrate, a difference which corresponds with the bumps height.

The flexural suspension limits the displacement range of the shuffle motor. The maximum displacement range is determined by the total in-plane stiffness of the flexural suspension and by the maximum output force of the motor. For our measurements, we have therefore used suspensions with two types of flexures: crab-like flexures with a low in-plane stiffness to enable a large displacement range; and guided-end beam flexures with a high in-plane stiffness to enable force measurements over a wide range of output forces. We have also built three types of shuffle motors (types I, II and III) with variations in the plate length and in the clamps areas. The main dimensions of the fabricated motors are given in Table 1.

Table 1. Main dimensions of the three types of shuffle motors.

|  |  | Shuffle motor |  |  |
| :--- | :---: | :---: | :---: | :---: |
| Parameter | Symbol | Type I | Type II | Type III |
| Overall dimensions $(\mu \mathrm{m} \times \mu \mathrm{m})$ | - | $290 \times 410$ | $290 \times 440$ | $290 \times 470$ |
| Clamp area $\left(\mu \mathrm{m}^{2}\right)$ | $A_{\mathrm{c}}$ | 38260 | 40340 | 42410 |
| Plate length $(\mu \mathrm{m})$ | $2 L$ | 180 | 208 | 236 |
| Plate width $(\mu \mathrm{m})$ | $w$ |  | 94 |  |
| Plate thickness $(\mu \mathrm{m})$ | $t$ |  | 1.15 |  |
| Gap plate $/$ substrate $(\mu \mathrm{m})$ | $g_{\mathrm{a}}$ |  | 1.76 |  |
| Bump height $(\mathrm{nm})$ | $b$ |  | 210 |  |
| Insulating layer thickness $(\mathrm{nm})$ | $t_{\mathrm{d}}$ |  | 210 |  |

## 4. Modeling

The theoretical model derived by Tas et al. predicts the performance of a shuffle motor operated in the stable region below the pull-in instability of the plate [8]. In this section, we derive a model to calculate the step size of a shuffle motor operated above the pull-in voltage. That is, the voltage above which the electrostatic attraction force between the plate and the substrate is greater than the restoring mechanical force, resulting in the collapse of the plate. We also present models for maximum force and operating voltage of such a motor.

The shuffle motor converts normal inflexion of an electrostatically actuated plate into an in-plane step. To evaluate the step size, we consider an elastic plate with length $2 L$, thickness $t$ and width $w$, as shown in Figure 5. The plate is suspended at a distance $g_{a}$ above the bottom electrode, which is covered with an insulating layer of thickness $t_{\mathrm{d}}$. The step size depends on the degree of the plate inflexion, which is directly related to the voltage $U_{\mathrm{p}}$ applied to the plate. Fine- and coarse-stepping motion can be attained by an appropriate choice of the actuation voltage. In the fine-stepping mode, the plate is operated in the stable regime below the pull-in voltage $U_{\mathrm{pi}}\left(U_{\mathrm{p}}<U_{\mathrm{pi}}\right)$. In this operational mode, the mechanical restoring force of the plate counteracts the electrostatic force, resulting in a relatively small normal inflexion of the plate and a fine longitudinal displacement $\delta_{\mathrm{f}}$. In the coarse-stepping mode, the actuation voltage is set above pull-in ( $U_{\mathrm{p}} \geq U_{\mathrm{pi}}$ ). In this case, the electrostatic attractive force between the plate and the substrate increases more rapidly with decreasing gap than the restoring mechanical force of the plate, causing instability and the ultimate collapse of the plate onto the substrate. The large deformation of the plate associated with this collapse results in a relatively large longitudinal displacement $\delta_{\mathrm{c}}$.

Figure 5. Schematic diagram showing the parameters used to model the plate deformation.


### 4.1. Fine-stepping mode

When a voltage $U_{\mathrm{p}}$ is applied on the plate, an electrostatic force inflects the plate towards the grounded substrate. This induces a longitudinal contraction of the plate, moving the axially unrestrained end of the plate closer to the fixed end. To estimate the longitudinal contraction $\delta_{\mathrm{f}}$, we consider a plate of length 2 L . From elastic bending theory, its longitudinal displacement $\delta_{\mathrm{f}}$, also called the curvature shortening, can be approximated by [23]:

$$
\begin{equation*}
\delta_{f}=\frac{1}{2} \int_{0}^{2 L}\left(\frac{d z}{d x}\right)^{2} d x \tag{1}
\end{equation*}
$$

where $z(x)$ is the inflexion curve of the plate. The inflexion curve of half of the plate is approximated with the function:

$$
\begin{equation*}
z(x)=\frac{1}{2} A\left(1-\cos \left(\frac{\pi x}{L}\right)\right) \text { for } 0 \leq x \leq L \tag{2}
\end{equation*}
$$

where $A$ is the center inflexion of the plate. The shape function $z(x)$ satisfies the boundary conditions $\left(z(0)=z^{\prime}(0)=z^{\prime}(L)=0\right.$ and $\left.z(L)=A\right)$, giving a reasonable approximation of the real inflexion profile. The longitudinal displacement $\delta_{\mathrm{f}}$ of the movable end of the plate, operated below the pull-in voltage, is obtained by substituting the assumed inflexion profile into (1):

$$
\begin{equation*}
\delta_{f}=\frac{\pi^{2}}{8} \frac{A^{2}}{L} \tag{3}
\end{equation*}
$$

This result is in good agreement with the one derived by [7].
The maximum inflexion of the plate at the midpoint before the pull-in instability is assumed to be $40 \%$ of the effective gap $g_{\text {eff }}$ [24] defined as:

$$
\begin{equation*}
g_{e f f}=g_{a}+\frac{t_{d}}{\varepsilon_{r}} \tag{4}
\end{equation*}
$$

where $\varepsilon_{\mathrm{r}}$ is the relative permittivity of the insulating layer.

### 4.2. Coarse-stepping mode

In the coarse-stepping mode, the plate is actuated above the pull-in voltage $U_{\text {pi }}$. The pull-in voltage is a function of the geometry of the plate and depends on the axial load $S$. For zero axial load, the pull-in voltage of the plate is given by [7]:

$$
\begin{equation*}
U_{P I}=\sqrt{0.75 \frac{E t^{3} g_{\text {eff }}^{3}}{\varepsilon_{o} L^{4}}} \tag{5}
\end{equation*}
$$

where $E$ is the plate's Young modulus and $\varepsilon_{0}$ is the permittivity of air. When the actuation voltage equals or exceeds the pull-in voltage $U_{\mathrm{p}} \geq U_{\mathrm{pi}}$, the plate makes physical contact with the grounded substrate, producing a relatively large longitudinal displacement, $\delta_{\mathrm{c}} \gg \delta_{\mathrm{f}}$. The displacement $\delta_{\mathrm{c}}$ corresponds to that induced by a plate of length $2 L_{\mathrm{f}}$ with a center inflexion of $A_{\text {max }}$ :

$$
\begin{equation*}
\delta_{c}=\frac{\pi^{2}}{8} \frac{A_{\max }^{2}}{L_{f}} \tag{6}
\end{equation*}
$$

$A_{\text {max }}$ is determined by the initial gap $g_{\mathrm{a}}$ between the plate and the substrate minus the height of bumps ( $A_{\max }=g_{\mathrm{a}}-b$ ). The length $2 L_{\mathrm{f}}$ is the part of the plate that is not in contact with the substrate and depends on the applied voltage $U_{\mathrm{p}}$. By increasing the plate voltage, the contact region $c$ increases, decreasing the free length. To determine the free length of the plate, we consider the inflected part of length $L_{\mathrm{f}}$, as illustrated in Figure 5. The electrostatic force $F_{\mathrm{e}}$ working on this part is given by:

$$
\begin{equation*}
F_{e}=\frac{1}{2} U_{p}^{2} \int_{0}^{L_{f}} \frac{\varepsilon_{o} w}{\left(g_{e f f}-z(x)\right)^{2}} d x \tag{7}
\end{equation*}
$$

The mechanical restoring force $F_{\mathrm{m}}$ of the inflected part can be approximated as the force required to inflect a cantilever plate of length $L_{\mathrm{f}}$ with the guided-end boundary condition at distance $A_{\text {max }}$ :

$$
\begin{equation*}
F_{m}=\frac{E t^{3} w}{L_{f}^{3}} A_{\max } \tag{8}
\end{equation*}
$$

The free length as a function of the applied voltage can be calculated by equating the electrostatic force and the mechanical restoring force $\left(F_{\mathrm{e}}=F_{\mathrm{m}}\right)$. It follows from (6) and (7) that the free length is independent of the original length $2 L$ of the plate. This implies that shuffle motors with different plate lengths will have the same step size for actuation voltages exceeding the pull-in voltage.

### 4.3. Output force

The shuffle motor employs inflexion and relaxation of the elastic plate for pulling and pushing the non-activated clamp, as illustrated in Figure 1. When a tensile load $S$ is applied to the clamp, an axial strain is induced in the plate, reducing the clamp displacement correspondingly. If the axial strain equals the inflexion-induced shortening $\delta$, the net displacement of the clamp will be zero. The axial force associated with this strain is the maximum force that can be pulled by the shuffle motor:

$$
\begin{equation*}
F_{\text {pull }}=\frac{E t w}{2 L} \delta \tag{9}
\end{equation*}
$$

Relaxation of the plate pushes the non-activated clamp. A compressive load $S$ greater than or equal to the buckling force of the plate will prevent stretching. The buckling load of the plate is thus the maximum load a shuffle motor can push. The buckling load of a rectangular plate clamped on the both ends is given by [23]:

$$
\begin{equation*}
F_{\text {push }}=\frac{\pi^{2}}{3} \frac{E t^{3} w}{(2 L)^{2}} \tag{10}
\end{equation*}
$$

Depending on the geometry of the plate, the maximum output force of the shuffle motor will be limited either by the pulling or by the pushing force.

In order to transmit the force produced by the plate, a sufficient clamp force is required. The normal force $F_{\mathrm{n}}$ is generated by applying a voltage difference $U_{\mathrm{c}}$ between the clamp and the grounded substrate:

$$
\begin{equation*}
F_{n}=\frac{1}{2} \frac{\varepsilon_{o} A_{c}}{\left(b+\frac{t_{d}}{\varepsilon_{r}}\right)^{2}} U_{c}^{2} \tag{11}
\end{equation*}
$$

where $A_{\mathrm{c}}$ is the clamping area. The normal force induces a frictional force $F_{\mathrm{t}}$ in the clamp, whose magnitude depends on the coefficient of friction, $\mu\left(F_{\mathrm{t}}=\mu F_{\mathrm{n}}\right)$. In order to prevent slipping of the clamps, the frictional force must be larger than the maximum output force of the motor. The coefficient of static friction between silicon nitride/silicon nitride contacts is between 0.55 and 0.85 [25]. Performance characteristics of the shuffle motors as predicted by the derived analytical models are summarized in Table 2. For these calculations, we have used the dimensions listed in Table 1. Furthermore we have used $E=160 \mathrm{GPa}$ as the Young's modulus of polysilicon, $\mu=0.55$ for the coefficient of friction, $\varepsilon_{0}=8.85 \times 10^{12} \mathrm{C}^{2} \cdot \mathrm{~N}^{-1} \cdot \mathrm{~m}^{-2}$ for the permittivity of air and $\varepsilon_{\mathrm{r}}=7.5$ for the relative permittivity of silicon nitride.

Table 2. Calculated performances of the shuffle motors.

|  |  | Shuffle motor |  |  |
| :--- | :---: | :---: | :---: | :---: |
| Parameter | Symbol | Type I | Type II | Type III |
| Pull-in voltage $(\mathrm{V})$ | $U_{\mathrm{pi}}$ | 42.2 | 31.6 | 24.5 |
| Fine stepping $(\mathrm{nm})$ | $\delta_{\mathrm{f}}$ | 7.0 | 6.0 | 5.3 |
| Coarse stepping $U_{\mathrm{pi}} \leq U_{\mathrm{p}} \leq U_{\mathrm{p}}(\mathrm{nm})$ | $\delta_{\mathrm{c}}$ | $48 \div 62$ | $42 \div 62$ | $37 \div 62$ |
| Maximum pull force $(\mathrm{mN})$ | $F_{\text {pull }}$ | 4.8 | 4.2 | 3.7 |
| Maximum push force $(\mathrm{mN})$ | $F_{\text {push }}$ | 2.3 | 1.7 | 1.3 |
| Clamp force $15 \mathrm{~V} \leq U_{\mathrm{c}} \leq 60 \mathrm{~V}(\mathrm{mN})$ | $F_{\mathrm{n}}$ | $0.7 \div 10.8$ | $0.7 \div 11.3$ | $0.7 \div 11.9$ |
| Frictional force $15 \mathrm{~V} \leq U_{\mathrm{c}} \leq 60 \mathrm{~V}(\mathrm{mN})$ | $F_{\mathrm{t}}$ | $0.4 \div 5.9$ | $0.4 \div 6.2$ | $0.4 \div 6.5$ |

## 5. Measurements

In this section, we report on measurements of speed, step size and output force performed on the fabricated shuffle motors. Unless otherwise noted, the measurement results apply to 'Type I' motors.

### 5.1. Stepping motion

We have successfully generated stepping motion with the shuffle motors using a coordinated sequence of clamps and plate voltages, as shown in Figure 4(d),(e). The voltage sequence was generated using a multichannel analog output card and a high speed voltage amplifier. After each operation cycle, the polarity of the actuation voltage was reversed in order to reduce charge accumulation in the insulating silicon nitride layer.

The positioning performance of the motors was measured using an image processing technique based on the Fast Fourier Transform (FFT) [26]. In this technique, the in plane motion of periodic patterns is recorded using a static video camera mounted on an optical microscope. The video showing the movement of the periodic patterns is then analyzed frame by frame in Matlab ${ }^{\circledR}$ using the FFT method. The phase differences between the video frames are used to determine the displacement of the periodic patterns. This measurement technique enables sub-pixel displacement measurements with a resolution of a few nanometers [27]. We have performed these measurements using the periodic patterns integrated on the shuffle motor (see Figure 4(a),(d),(e)).

Figure 6 shows typical measurements of the motor position as a function of time. In Figure 6(a), the motor was actuated at a cycling frequency of 1 Hz and a driving voltage of 45 V for both the clamps and plate. Due to the high resolution of these measurements, the individual steps can be clearly resolved. The step size was found to be 50 nm , which is in good agreement with the theoretical prediction (see Table 2). The maximum displacement range of the motor was limited to $\pm 70 \mu \mathrm{~m}$ by the suspension design. To achieve this relatively large stroke, we have used a motor suspended by a rather compliant crab-like suspension. Figure 6(b) shows the motor displacement versus time spanning the entire displacement range in a single direction.

Figure 6. Measured displacement of a 'Type I' shuffle motor as a function of time. (a) Data were taken at a stepping frequency of 1 Hz and a driving voltage of 45 V for both the clamps and plate; (b) Data were taken at a stepping frequency of 200 Hz and voltages of 30 V and 50 V for the clamps and plate, respectively. The measurements are for one direction only. The displacement range is double.


### 5.2. Speed

The speed of the motor was determined by measuring the distance traveled for a given driving frequency and a fixed number of steps (typically 100 steps) [10]. For each frequency, we took the average value of three measurements. The average deviation for each measurement point was lower than $0.1 \%$. Figure 7 shows the speed of the shuffle motor as a function of the stepping frequency. The data were taken at frequencies up to 80 kHz with both clamps and plate driven at 45 V . The maximum frequency range of 80 kHz was limited by the driving electronics. These data show that the velocity increases nearly linearly with the cycling frequency over the measured range, with a maximum velocity of $3.6 \mathrm{~mm} / \mathrm{s}$ at 80 kHz . The slope of the speed curve gives a measure for the average step size, which is about 50 nm . Considering that the plate resonant frequency was calculated to be 300 kHz , a considerably larger cycling frequency and speed is expected by improving the driving electronics.

Figure 7. Measured speed of a 'Type I' shuffle motor versus the stepping frequency with clamps and plate driven at 45 V .


### 5.3. Step size

The velocity of the shuffle motor depends on the cycling frequency and on the average step size. As discussed in Sections 4.1 and 4.2, the step size of the motor is a function of the degree of plate deformation, which is directly related to the plate actuation voltage. As shown in Figure 8, the two operational modes and the pull-in instability can be clearly identified. These measurements were performed at a cycling frequency of 200 Hz and a clamping voltage of 30 V . The pull-in voltage is found to be $\sim 39 \mathrm{~V}$, which agrees reasonably with the calculated 42.2 V (see Table 2). We have observed an average step size from $0.6-7.0 \mathrm{~nm}$ for the motor operated below the pull-in voltage. Operation of the plate above the pull-in instability causes the plate to collapse on the substrate, resulting in a large plate deformation and a significant increase in step size. In the coarse-stepping
operational mode, we have measured adjustable nanometer resolution steps ranging between 49 nm and 62 nm , corresponding to plate voltages varying between 40 V and 70 V . The measured step size for both non-contact and contact operational modes are in fair agreement with the theoretical predictions listed in Table 2.

Figure 8. Measured average step size of a 'Type I' shuffle motor versus the plate actuation voltage. A cycling frequency of 200 Hz and a clamp voltage of 30 V were used for these measurements.


The measurements of the average step size for the three types of shuffle motors operated above pull-in are shown in Figure 9 and compared with the analytical curve. The measured step size is in good agreement with the theoretical prediction and does not depend on the length of the plate, as expected from the model (see Section 4.2).

As shown in Figure 10, the step size of a shuffle motor is a function of the load applied to the clamps. To investigate this dependence, we have suspended the motor with connection springs on both clamps, as schematically illustrated in the inset of Figure 10, and we have measured the motor's displacement as a function of time at a fixed cycling frequency. From these data, we have extracted the step size as a function of the displacement and related it to the mechanical restoring force applied by the connection springs. When the motor walked against the springs in the forward motion, the step size decreased from 53 nm at zero load to 9 nm at 1.5 mN load. Conversely, when the motor traveled backward, the spring force assisted the motion, increasing the step size to 88 nm at a load of 1.5 mN . Figure 10 shows that the step size varies linearly with the applied load.

Figure 9. Average step size of the fabricated shuffle motors, operated above the pull-in instability. A cycling frequency of 200 Hz and a clamp voltage of 30 V were used for these measurements.


Figure 10. Measured step size of a 'Type I' shuffle motor versus the load on the clamps. The data were taken at a cycling frequency of 200 Hz with clamps and plate voltages of 55 V .


### 5.4. Output force

To determine the output force of the motors as a function of the driving voltage, we have measured the maximum displacement range while varying the voltages on the clamps and plate. The output force
was deduced from the displacement measurement and from the calculated stiffness of the suspension. Stiff suspensions were employed to enable force measurements over a wide range of output forces. The shuffle motors used for these measurements were suspended by fixed-guided beam flexures attached on both clamps, such as the one visible in Figure 4(a). The stiffness of the beam elements is non-linear for large displacements due to the stress stiffening effect [24]. The stiffness-deflection function was calculated by finite element method using ANSYS software. Since the motors worked against half of the connection springs at any given time (i.e., the activated clamp balanced the force of the connection springs attached to it), only one half of the total suspension stiffness was used in the calculation. Either in the pull or in the push mode, the spring force can limit the maximum displacement of the motor. In the pull mode, the displacement is stopped when the tensile load on a non-activated clamp is sufficient to raise the pull-in voltage above the driving voltage. In the push mode, the maximum displacement is reached when the compressive load exceeds the plate buckling force. The measurements were performed on all of the three types of shuffle motors, using clamp voltages varying from 15 V to 60 V . The motors were operated above the pull-in instability. Figure 11 shows the output force as a function of the clamp voltage for the three types of shuffle motors, operated at different plate voltages.

Figure 11. Measured output force of the shuffle motors versus the load on the clamps. The symbols $\bullet, \mathbf{\Delta}$ and $\star$ refer to the shuffle motors I, II and III, respectively (see legend in Figure 9). The data were taken at a cycling frequency of 200 Hz .


The output force characteristics are qualitatively similar for the three types of motors. At lower plate voltages, the tensile load induced by the suspension limits the output force of the motor. The tensile load on a non-activated clamp, which increases monotonically with the displacement of the motor, raises the pull-in voltage of the plate. At a given displacement, the tensile load becomes sufficiently high enough to raise the pull-in voltage above the driving voltage, preventing further motion of the motor. This is visible through the different plateaus in the curves. By increasing the plate voltage, the output force increases until it saturates at another maximum value, which is given by the buckling force of the plate.

The maximum output force of 'Type III' motor is 1.1 mN ( $\star$ symbols in Figure 11), which is in fair agreement with the theoretical value of 1.3 mN estimated for $F_{\text {push }}$ (see Table 2). Using (10) for the buckling force in combination with the measured output force $\left(F_{\text {III }}\right)$ and the length of the plate $\left(L_{\text {III }}\right)$ of 'Type III' motor, the output force ( $F_{\text {II }}$ ) of 'Type II' motor can be accurately predicted:

$$
\begin{equation*}
F_{I I}=F_{I I I}\left(\frac{L_{I I}}{L_{I I}}\right)^{2}=1.1\left(\frac{236}{208}\right)^{2}=1.4 m N \tag{12}
\end{equation*}
$$

This prediction is in good agreement with the measured force of 1.4 mN , as shown in Figure 11 ( $\boldsymbol{\Delta}$ symbols). The same calculation for 'Type I' motor with a plate length of $180 \mu \mathrm{~m}$ predicts a maximum output force of 1.9 mN , which is close to the measured force of 1.7 mN , as shown in Figure 11 (- symbols).

These results are in fair agreement with the model developed in Section 4.3. Indeed, the maximum output force of the shuffle motor is determined by the lower of the pulling or the pushing force. Hence, in our design, the output force is in all cases limited by the pushing force. Considering this, the measured output forces are lower than the estimated values for $F_{\text {push }}$. The small discrepancy can be attributed to the fact that the plates are actually perforated; that is, the buckling forces of the plates are lowered due to the presence of bumps.

The maximum output force of 1.7 mN demonstrated for 'Type I' shuffle motor was achieved with actuation voltages of 55 V on both clamps and plate. This corresponds to a force density of $4.78 \mathrm{~N} \cdot \mathrm{~m}^{-2} \cdot \mathrm{~V}^{-2}$ and a power density of $1.7 \times 10^{-2} \mathrm{~W} \cdot \mathrm{~m}^{-2} \cdot \mathrm{~V}^{-2}$, which are the highest values reported so far for electrostatic microactuators (see Figure 1 and Table A1).

### 5.5. Lifetime

A limited lifetime test was conducted to determine the durability of the shuffle motor. The motor was run continuously for five days at a cycling frequency of 80 kHz , generating nearly 34 billion steps, equivalent to a displacement of 1.5 km . This did not result in any noticeable change in performance.

## 6. Conclusions

We have presented a high-performance bidirectional electrostatic linear stepper micromotor characterized by a small size ( $<1 \mathrm{~mm}^{2}$ ), large output forces ( mN range), large strokes (hundreds of $\mu \mathrm{m}$ ), high resolution ( nm range), high speed ( $\mathrm{mm} / \mathrm{s}$ range), low power consumption (few $\mu \mathrm{W}$ ) and high durability. The performance goals for our microactuator were motivated by its potential use in a magnetic memory based on scanning probe technology, the Micro Scanning Probe Array Memory
( $\mu$ SPAM) [28]. Besides data storage systems, the demonstrated performance makes our motor a promising candidate for diverse MEMS applications, including optical systems (microscopy) and robotics (microassembly).

## Acknowledgements

The authors thank M. de Boer, R. Sanders, T. Lammerink and MESA+ clean room staff for their contribution to the fabrication and measurements. Part of the research presented in this paper was performed at the CIRMM, IIS, the University of Tokyo, Japan. This work was part of the research program on Micro Scanning Probe Array Memory ( $\mu$ SPAM) supported by the Dutch Technology Foundation (STW).

## References

1. Fujita, H. Microactuators and micromachines. Proc. IEEE 1998, 86, 1721-1732.
2. Thielicke, E.; Obermeier, E. Microactuators and their technologies. Mechatronics 2000, 10, 431-455.
3. Trimmer, W.S.N. Microrobots and micromechanical systems. Sens. Actuator. 1989, 19, 267-287.
4. Fearing, R.S. Powering 3 dimensional microrobots: Power density limitations. In Proceedings of the IEEE International Conference on Robotics and Automation (ICRA)—Tutorial on Micro Mechatronics and Micro Robotics, Leuven, Belgium, May 1998; pp. 1-15.
5. Yeh, R.; Kruglick, E.J.J.; Pister. K.S.J. Surface-micromachined components for articulated microrobots. J. Microelectromech. Syst. 1996, 5, 10-17.
6. Baltzer, M.; Kraus, T.; Obermeier, E. Linear stepping actuator in surface micromachining technology for low voltages and large displacements. In Proceedings of the International Conference on Solid-State Sensors and Actuators (Transducers), Chicago, IL, USA, June 1997; pp. 781-784.
7. Tas, N.R.; Sonnenberg, A.H.; Sander, A.F.M.; Elwenspoek, M.C. Surface micromachined linear electrostatic stepper motor. In Proceedings of the 10th IEEE Annual International Workshop on Micro Electro Mechanical Systems (MEMS), Nagoya, Japan, January 1997; pp. 215-220.
8. Tas, N.; Wissink, J.; Sander, L.; Lammerink, T.; Elwenspoek, M. Modeling, design and testing of the electrostatic shuffle motor. Sens. Actuator. A Phys. 1998, 70, 171-178.
9. Yeh, R.; Hollar, S.; Pister, K.S.J. Single mask, large force, and large displacement electrostatic linear inchworm motors. J. Microelectromech. Syst. 2002, 11, 330-336.
10. de Boer, M.P.; Luck, D.L.; Ashurst, W.R.; Maboudian, R.; Corwin, A.D.; Walraven, J.A.; Redmond, J.M. High-performance surface-micromachined inchworm actuator. J. Microelectromech. Syst. 2004, 13, 63-74.
11. Kim, S.H.; Hwang, I.H.; Jo, K.W.; Yoon, E.S.; Lee, J.H. High-resolution inchworm linear motor based on electrostatic twisting microactuators. J. Micromech. Microeng. 2005, 15, 1674-1682.
12. Sarajlic, E.; Berenschot, E.; Tas, N.; Fujita, H.; Krijnen, G.; Elwenspoek, M. Fabrication and characterization of an electrostatic contraction beams micromotor. In Proceedings of the 19th IEEE International Conference on Micro Electro Mechanical Systems (MEMS), Istanbul, Turkey, January 2006, pp. 814-817.
13. Erismis, M.A.; Neves, H.P.; Puers, R.; van Hoof, C. A low-voltage large-displacement large-force inchworm actuator. J. Microelectromech. Syst. 2008, 17, 1294-1301.
14. Akiyama, T.; Shono, K. Controlled stepwise motion in polysilicon microstructures. J. Microelectromech. Syst. 1993, 2, 106-110.
15. Akiyama, T.; Collard, D.; Fujita, H. Scratch drive actuator with mechanical links for self-assembly of three-dimensional MEMS. J. Microelectromech. Syst. 1997, 6, 10-17.
16. Li, L.; Brown, J.G.; Uttamchandani, D. Study of scratch drive actuator force characteristics. J. Micromech. Microeng. 2002, 12, 736-741.
17. Mita, M.; Arai, M.; Tensaka, S.; Kobayashi, D.; Fujita, H. A micromachined impact microactuator driven by electrostatic force. J. Microelectromech. Syst. 2003, 12, 37-41.
18. Sarajlic, E.; Yamahata, C.; Cordero, M.; Fujita, H. An electrostatic 3-phase linear stepper motor fabricated by vertical trench isolation technology. J. Micromech. Microeng. 2009, 19, 074001.
19. Sarajlic, E.; Yamahata, C.; Cordero, M.; Fujita, H. Electrostatic rotary stepper micromotor for kew angle compensation in hard disk drives. In Proceedings of the 22nd IEEE Intertional Conference on Micro Electro Mechanical Systems (MEMS), Sorrento, Italy, January 2009, pp. 1079-1082.
20. Rodgers, M.S.; Sniegowski, J.J. 5-level polysilicon surface micromachine technology: Application to complex mechanical systems. In Proceedings of the IEEE Solid State Sensors and Actuators Workshop, Hilton Head, SC, USA, June 1998; pp. 144-149.
21. Sarajlic, E.; Berenschot, E.; Krijnen, G.; Elwenspoek, M. Versatile trench isolation technology for the fabrication of microactuators. Microelectronic Eng. 2003, 67-68, 430-437.
22. Sarajlic, E.; Berenschot, E.; Fujita, H.; Krijnen, G.; Elwenspoek, M. Bidirectional electrostatic linear shuffle motor with two degrees of freedom. In Proceedings of the 18th IEEE International Conference on Micro Electro Mechanical Systems (MEMS), Miami, FL, USA, January 2005; pp. 391-394.
23. Gere, J.M.; Timoshenko, S.P. Mechanics of materials; Chapman \& Hall: London, UK, 1991.
24. Pamidighantam, S.; Puers, R.; Baert, K.; Tilmans, H.A.C. Pull-in voltage analysis of electrostatically actuated beam structures with fixed-fixed and fixed-free end conditions. J. Micromech. Microeng. 2002, 12, 458-464.
25. Deng, K.; Ko, W.H. A study of static friction between silicon and silicon compounds. J. Micromech. Microeng. 1992, 2, 14-20.
26. Vanapalli, S. Techniques for characterization of in-plane displacement for microactuators. Master Thesis, Department of Electrical Engineering, University of Twente, The Netherlands, 2004.
27. Yamahata, C.; Sarajlic, E.; Jalabert, L.; Kumemura, M.; Collard, D.; Fujita, H. Mechanical characterization of biomolecules in liquid using silicon tweezers with subnanonewton resolution. In Proceedings of the 22nd IEEE International Conference on Micro Electro Mechanical Systems (MEMS), Sorrento, Italy, January 2009; pp. 607-610.
28. Bolks, M.; Hanssen, F.; Abelmann, L.; Havinga, P.; Hartel, P.; Jansen, P.; Lodder, C.; Smit, G. Micro scanning probe array memory ( $\mu \mathrm{SPAM}$ ). In Proceedings of the 2nd Progress Workshop on Embedded Systems, Veldhoven, The Netherlands, October 2001; pp. 17-26.

## Appendix

Table A1. Micromachined electrostatic linear micromotors with large output force.

| Motor | A | B | C | D | E | F | G | H | I |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Authors | Sarajlic et al. |  | Sarajlic et al. |  | Yehet al. | Akiyama et al. | Erismis et al. | Tas et al. | Kim et al. |
| Reference | This work | [16] | [12] | [10] | [9] | [15] | [13] | [8] | [11] |
| Overall dimensions ( $\mu \mathrm{m} \times \mu \mathrm{m}$ ) | $290 \times 410$ | $60 \times 75$ | $290 \times 470$ | $200 \times 1500$ | $1500 \times 2000$ | $60 \times 75$ | $1800 \times 3700$ | $400 \times 560$ | $2200 \times 2800$ |
| Voltages $U_{\mathrm{c}} / U_{\mathrm{p}}(\mathrm{V} / \mathrm{V})$ | $55 / 55$ | $200 / 200$ | $50 / 60$ | 150 / 65 | $33 / 33$ | 112 / 112 | $10 / 10$ | $40 / 25$ | 19/15 |
| Driving Frequency (kHz) | 80 | 10 | 10 | 80 | 1 | 0.1 | 1 | 1.16 | 1.68 |
| Speed (mm/s) | 3.6 | 0.25 | 0.1 | 3 | 4 | 0.008 | 1 | 0.1 | 0.003 |
| Displacement range ( $\mu \mathrm{m}$ ) | 140 | 60 | 140 | 200 | 80 | 100 | 24 | 43 | 3 |
| Average step size (nm) | 45 | 25 | 10 | 37.5 | 4000 | 80 | 1000 | 85 | 10 |
| Dynamic range (-) | 3100 | 2400 | 14000 | 5330 | 20 | 1250 | 24 | 500 | 300 |
| Output force (mN) | 1.72 | 0.25 | 0.49 | 0.45 | 0.26 | 0.063 | 0.05 | 0.043 | 0.013 |
| Normalized force density $\left(\mathrm{N} \cdot \mathrm{~m}^{-2} \cdot \mathrm{~V}^{-2}\right)$ | 4.78 | 1.39 | 1.19 | 0.13 | 0.08 | 1.12 | 0.075 | 0.18 | 0.007 |
| Power (W) | $6.2 \times 10^{-6}$ | $6.3 \times 10^{-8}$ | $4.9 \times 10^{-8}$ | $1.4 \times 10^{-6}$ | $1.0 \times 10^{-6}$ | $5.0 \times 10^{-10}$ | $5.0 \times 10^{-8}$ | $4.3 \times 10^{-9}$ | $3.8 \times 10^{-11}$ |
| Normalized power density $\left(\mathrm{W} \cdot \mathrm{~m}^{-2} \cdot \mathrm{~V}^{-2}\right)$ | $1.7 \times 10^{-2}$ | $3.5 \times 10^{-4}$ | $1.2 \times 10^{-4}$ | $3.9 \times 10^{-4}$ | $3.2 \times 10^{-4}$ | $8.9 \times 10^{-6}$ | $7.5 \times 10^{-5}$ | $1.8 \times 10^{-5}$ | $2.2 \times 10^{-8}$ |

© 2010 by the authors; licensee MDPI, Basel, Switzerland. This article is an Open Access article distributed under the terms and conditions of the Creative Commons Attribution license (http://creativecommons.org/licenses/by/3.0/).

