Glottal flow through a two-mass model: Comparison of Navier–Stokes solutions with simplified models

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Chapter 3

Glottal flow through a two-mass model: comparison of Navier-Stokes solutions with simplified models

M.P. de Vries, H.K. Schutte, A.E.P. Veldman, G.J. Verkerke

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Abstract
A new numerical model of the vocal folds is presented based on the well-known
two-mass models of the vocal folds. The two-mass model is coupled to a model of
glottal airflow based on the incompressible Navier-Stokes equations. Glottal waves
are produced using different initial glottal gaps and different subglottal pressures.
Fundamental frequency, glottal peak flow and closed phase of the glottal waves
have been compared with values known from the literature. The phonation
threshold pressure was determined for different initial glottal gaps. The phonation
threshold pressure obtained using the flow model with Navier-Stokes equations
corresponds better to values determined in normal phonation than the phonation
threshold pressure obtained using the flow model based on the Bernoulli equation.
Using the Navier-Stokes equations, an increase of the subglottal pressure causes
an increase of both the fundamental frequency and the glottal peak flow, whereas
the fundamental frequency in the Bernoulli-based model remains constant with
increasing pressure.

List of symbols
\( CQ \) closed quotient
\[ F \] vector representing external forces on the fluid
\( F_0 \) fundamental frequency
\( U_g \) glottal peak flow
\( p \) pressure
\( p_s \) subglottal pressure
\( u \) vector with the velocity components
\( \nu \) kinematic viscosity of the fluid
\( u \) horizontal velocity of fluid
\( v \) vertical velocity of fluid
\( m_1 \) lower mass
\( k_1 \) lower spring stiffness
\( r_1 \) lower damper
\( m_2 \) upper mass
\( k_2 \) upper spring stiffness
\( r_2 \) upper damper
\( k_c \) coupling stiffness
\( k_{col1} \) lower collision spring stiffness
\( k_{col2} \) upper collision spring stiffness
\( x_1 \) deflection of \( m_1 \)
\( x_2 \) deflection of \( m_2 \)
\( \zeta_1 \) lower damping ratio
\( \zeta_2 \) upper damping ratio
3.1 Introduction

To understand the process of voice production, several authors have investigated the pressure-flow relationship in the glottis (e.g., Van den Berg et al., 1957; Scherer and Titze, 1983; Alipour et al., 1996a; Guo and Scherer, 1993; Liljencrants, 1991). These examinations were all performed in a static glottis. Other investigations include the dynamic behavior of the voice source, which has been experimentally studied by Shadle et al. (1999) and Mongeau et al. (1997). Numerical modeling of the interaction between the oscillating vocal folds and the airflow in the glottis is performed using a simplified description of the vocal fold combined with a simplified description of the airflow (e.g., Ishizaka and Flanagan, 1972; Pelorson et al., 1994; Story and Titze, 1995; Herzel et al., 1995; Lous et al., 1998). With these so-called lumped parameter models, glottal waves are produced as a result of a flow-induced oscillation of the vocal folds. Generally, in these lumped parameter models, the pressure and flow in the glottis are related by the Bernoulli equation. This implies that certain assumptions concerning the physical characteristics of the fluid, air in our case, have to be made: the flow is assumed to be steady, laminar, non-viscous, and incompressible. In a cross-section, pressure and velocity are usually assumed to be constant, although in reality, pressure and velocity vary over a cross-section. The velocity also has a transverse component that is ignored. The assumptions mentioned before, result in an inaccurate description of the flow. To improve the accuracy of the flow calculations, one has to resort to the two-dimensional Navier-Stokes equations, which describe the non-steady and viscous behavior of a fluid under an external load. In this study, Navier-Stokes equations for incompressible flow are implemented.

Although solving the Navier-Stokes equations is very time-consuming, today’s computing power ensures acceptable calculation times. In the last decade, the Navier-Stokes equations already have been used in research concerning voice production. Alipour et al. (1996a), Guo and Scherer (1993), and Liljencrants (1991) presented data using the Navier-Stokes equations in a static model of the vocal folds. The pressure-flow relationships resulting from these investigations in the static glottis are related to the dynamic situation during phonation.

Recently, Alipour and Titze (1996b) combined the Navier-Stokes equations for incompressible flow with a dynamic Finite Element Method (FEM) model of the vocal folds and simulated phonation in this way. In their model, the finite element method model of the vocal fold and the model of the Navier-Stokes equations exchange information, resulting in glottal waves. A disadvantage of their model is that only a subglottal flow rate can be prescribed instead of an initial pressure. An initial Bernoulli solution is needed in their model to approximate the flow rate that occurs at physiological subglottal pressures.

When the lumped parameter models of the vocal folds (Ishizaka and Flanagan, 1972; Herzel et al., 1995; Pelorson et al., 1994) are compared with the model of Alipour and Titze (1996b), it can be seen that the glottal airflow description, based on the Bernoulli equation in the lumped parameter models, is replaced by the
Navier-Stokes equations in the Alipour and Titze model, and the description of the vocal folds by a number of masses, springs and dampers in the lumped parameter models is replaced by a FEM model. In the present study we will present a model composed of a lumped parameter model of the vocal folds combined with a Navier-Stokes model of the glottal flow. In this way, the need to do assumptions concerning the viscous losses, as is needed in studies using a Bernoulli-based equation, is not necessary. In the Navier-Stokes equations, viscous effects are included. We will give a comparison between results obtained using the Navier-Stokes equations versus results obtained using the Bernoulli-based model. For the model of the vocal fold, we use a lumped parameter model, i.e. the two-mass model. In this way, only one of the two steps presented by Alipour and Titze (1996b) has been performed. This study is meant to be a step between the low-order models (the lumped parameter models) and the high order models. Also a new model of the aerodynamics is presented, in which it is possible to prescribe the subglottal pressure.

![Two-mass model](image)

3.2 Materials and Methods

A model of a vibrating vocal fold has been developed, composed of a two-mass model describing the vocal fold and the Navier-Stokes equations describing the glottal flow. Due to symmetry, only one vocal fold is considered. First the two-mass model simulating the vocal fold will be described, followed by the Navier-Stokes equations describing the aerodynamics. The interaction between the two-mass model and the Navier-Stokes equations will be described in a following section.
### TABLE I. Parameter values for the two mass model: Ishizaka and Flanagan parameters and De Vries parameters

<table>
<thead>
<tr>
<th></th>
<th>I&amp;F parameters</th>
<th>De Vries parameters</th>
</tr>
</thead>
<tbody>
<tr>
<td>$M_1$ lower mass (g)</td>
<td>0.125</td>
<td>0.024</td>
</tr>
<tr>
<td>$M_2$ upper mass (g)</td>
<td>0.025</td>
<td>0.020</td>
</tr>
<tr>
<td>$k_1$ lower spring stiffness (N/m)</td>
<td>80</td>
<td>22</td>
</tr>
<tr>
<td>$k_2$ upper spring stiffness (N/m)</td>
<td>8</td>
<td>14</td>
</tr>
<tr>
<td>$k_c$ coupling spring stiffness (N/m)</td>
<td>25</td>
<td>10</td>
</tr>
<tr>
<td>$\zeta_1$ damping ratio (g/s)</td>
<td>0.1</td>
<td></td>
</tr>
<tr>
<td>$\zeta_2$ damping ratio (g/s)</td>
<td>0.6</td>
<td></td>
</tr>
<tr>
<td>$l_g$ glottal length (cm)</td>
<td>1.3</td>
<td></td>
</tr>
<tr>
<td>$k_{col1}$ collision spring stiffness (N/m)</td>
<td>$3k_1$</td>
<td></td>
</tr>
<tr>
<td>$k_{col2}$ collision spring stiffness (N/m)</td>
<td>$3k_2$</td>
<td></td>
</tr>
</tbody>
</table>

#### 3.2.1 Two-mass model of the vocal folds

A two-mass model simulates one vocal fold by means of two coupled oscillators (e.g., Ishizaka and Flanagan, 1972; Herzel et al., 1995; Pelorson et al., 1994; Lous et al., 1998). Each oscillator consists of a mass, a spring, and a damper (Fig. 1). Mass $m_1$, spring stiffness $k_1$, and damper $r_1$ represent the lower part of the vocal fold. Mass $m_2$, spring stiffness $k_2$, and damper $r_2$ represent the upper part. The two masses are coupled by a spring with stiffness $k_c$. The two masses, $m_1$ and $m_2$, are free to move only in a lateral direction. The deflections of $m_1$ and $m_2$ are $x_1$ and $x_2$, respectively. In the two-mass model, symmetry with respect to a plane parallel to the main flow axis is assumed, therefore it is sufficient to consider only one vocal fold. When the vocal fold approaches the symmetry line within a very short distance, collision springs with stiffness $k_{col1}$ and $k_{col2}$ are activated and have an influence on the masses $m_1$ and $m_2$, respectively, in the contralateral direction (e.g., Ishizaka and Flanagan, 1972; Pelorson et al., 1994). In this way, the effective spring stiffness during collision changes. As a consequence of this way of
modeling the collision, the glottal opening is allowed to have a small negative value.

In this study, we use two sets of values of the masses and springs. The first set of values, used earlier by Ishizaka and Flanagan (1972), has been applied also by several other researchers (e.g. Herzel et al., 1995; Steinecke and Herzel, 1995). The second set of values was proposed by De Vries et al. (1999). This new set of parameters is based on a finite element method (FEM) study of the mechanical behavior of a vocal fold. The values of the masses and springs are substantially smaller than the values used in previous studies. Both sets of parameter values are listed in Table I.

3.2.2 Aerodynamics

To obtain oscillation of the vocal folds, aerodynamic forces have to act on the two masses. In the present study, the aerodynamic forces result from the pressure distribution along the glottal surface as determined by the Navier-Stokes equations. As mentioned earlier, the incompressible two-dimensional Navier-Stokes equations, which are based on two conservation laws, are used:

1. Conservation of mass

\[ \nabla \cdot \mathbf{u} = 0 \]  \hspace{1cm} (1)

where \( \mathbf{u} \) is the vector with the velocity components.

2. Conservation of momentum

\[ \frac{\partial \mathbf{u}}{\partial t} + \nabla p = \mathbf{R} \]  \hspace{1cm} (2)

with

\[ \mathbf{R} = - (\mathbf{u} \cdot \nabla) \mathbf{u} + \nu (\nabla \cdot \nabla) \mathbf{u} + \mathbf{F} \]

In equation 2, \( p \) is the pressure, \( \nu \) is the kinematic viscosity of the fluid, and \( \mathbf{F} \) is the vector representing external forces on the fluid.

The term \( \frac{\partial \mathbf{u}}{\partial t} \) in equation 2 is discretized in time with a forward Euler method using time step \( \partial t \). This results in:

\[ \nabla \cdot \mathbf{u}^{n+1} = 0 \]  \hspace{1cm} (3)
\[ \frac{u^{n+1} - u^n}{\partial t} + \nabla p^{n+1} = R^n \]  

(4)

where \( n+1 \) is the new time step and \( n \) is the present time step. These terms can be rearranged to

\[ u^{n+1} = u^n + \partial_t R^n - \partial_t \nabla p^{n+1} \]  

(5)

This can be substituted into eq. 3, which results in the Poisson equation for the pressure:

\[ (\nabla \cdot \nabla) p^{n+1} = \nabla (\frac{u^n}{\partial_t} + R^n) \]  

(6)

Spatial discretization of the Navier-Stokes equations is performed in a Cartesian grid. This grid can be refined at places of particular interest. The three degrees of freedom of each cell in the grid are the velocity in the direction of the main flow \( u \), the velocity perpendicular to the main flow \( v \) and the pressure \( p \). The velocity in the direction of the main flow \( u \) is defined on the right edge of a cell, the velocity perpendicular to the main flow \( v \) is defined on the upper edge of a cell, and the pressure \( p \) is defined in the center of a cell. This staggered way of placing the variables is known as the Marker-And-Cell (MAC) method (Harlow and Welsh, 1965). The numerical advantage of this method is the uniqueness of the pressure.

The equations stated above have to be completed by boundary conditions. At the laryngeal wall, the tangential velocity is set to zero, simulating the sticking of the fluid to the wall, the so-called no-slip condition. The perpendicular velocity is set to zero, simulating the impermeability of the wall. At the inlet, the subglottal pressure is prescribed. The outlet conditions combine a prescribed pressure with normal derivatives (to the outlet surface) of the velocities set to zero. Since in the two-mass model only one vocal fold is modeled because of the symmetry along the glottis, the aerodynamics has to be symmetric along the glottis as well. At the symmetry line, the velocity perpendicular to the symmetry line is set to zero and the derivative of the axial velocity to its perpendicular coordinate is set to zero also.

The model of the aerodynamics makes use of adaptive time-steps: within every time step, a routine checks whether the time step is small enough to give reliable results. The routine tries to calculate for the optimal time step for the present situation. This adaptive time step is favourable in the present case because the aerodynamic grid changes during the simulation as a consequence of the moving vocal fold. In this way, a stable and fast solution is obtained. During the simulations, the time step varied between 32.0e-6 second and 0.5e-6 second. An thorough validation of an expanded version of the model of the aerodynamics is presented by Veldman et al. (1999).

To be able to compare the results using Navier-Stokes equations with results of simplified models, also simulations with a glottal flow model based on Bernoulliare
performed. The model used for comparison is presented before by Lous et al. (1998).

3.2.3 Interaction

The interaction between the vocal fold and the aerodynamic load takes place at the surface defined by the location of the two masses (Fig. 2). The masses are connected by rigid, massless plates. This configuration is also used by Lous et al. (1998), where they describe a two-mass model that uses the Bernoulli equation. In this configuration, sharp edges at the locations of the masses are present.

![Fig. 2. Nonuniform grid with model of the vocal fold](image)

The interaction between the air and the structure occurs in the grid of the aerodynamics. The cells of the grid in which a part of the two-mass model is present are supposed to be filled by a solid material. In this way, a distinction is made between cells that are filled with air (aerodynamic cells) and cells that are not filled with air (mechanic cells). In the aerodynamic cells, the Navier-Stokes equations are calculated. The mechanic cells provide the boundary conditions for the aerodynamic cells. In the grid, information concerning the pressure of the airflow is transferred to the two-mass system, and information about the position and the velocity of the two masses is transferred to the aerodynamic model in every time step.

The pressure distribution acting on the vocal fold surface has to be translated to two point forces that act on the two masses. These forces are not directly available from the aerodynamic model, but are derived from the pressures in the aerodynamic cells adjacent to the vocal fold. The pressure values of the cells that contain air and are adjacent to the cells that contain a part of the vocal fold, are multiplied by the area of the concerning cell. The resulting forces are calculated in
all fluid cells adjacent to the vocal fold cells. Finally, these forces are distributed over the two masses in such a way that they form a statically equivalent system.

The resulting new positions and velocities of the two masses are calculated and transferred to the aerodynamic model by defining a new distribution of aerodynamic cells and mechanic cells in the grid. The velocities of the mechanic cells at the glottal surface are calculated by interpolation of the velocities of the two masses. In this way, a dynamic boundary condition for the aerodynamic cells is defined, which is used in the next time step of the calculations of the aerodynamic forces. In this way, a realistic and time-accurate description of the interaction is obtained.

Because the two-mass model is by definition a two-dimensional model, the aerodynamics is also considered to be two-dimensional. To obtain results as glottal flow and aerodynamic forces, the third dimension is simulated by assuming a uniform distribution of the aerodynamic quantities along the length of the vocal fold. According to measurements performed by Baer (1981), the length of the vocal fold is modeled by a length of 1.3 cm. In this way, boundary effects that occur at the anterior and posterior glottal commisure are neglected.

To determine the numerical validity of the model, the number of cells has been varied until a grid was obtained for which a doubling of the number of cells does not result in a noticeable difference in the glottal waves. The grid is made non-uniform by taking smaller cells in the neighbourhood of the glottis because in this region larger velocity gradients and pressure gradients can be expected.

To obtain glottal waves, different values for the subglottal pressure $P_s$ were applied. In a previous study (De Vries et al., 1999), the properties of glottal waves that are produced at a pressure of 6 cm H$_2$O were compared to normal values of these properties. We also used the value of 6 cm H$_2$O in this study, which is chosen after Holmerg et al. (1989). They derived average values for several quantities concerning phonation in male and female. The normal value of the subglottal pressure of 4.3 cm H$_2$O, determined by Schutte (1980), has not been used because no glottal waves were obtained at that pressure for the specific set of model parameters used in our study. The properties of the glottal waves produced with the presented numerical model will be considered at $P_s=0.6$ kPa, which is almost equal to 6 cm H$_2$O. In this way, the properties of the glottal waves obtained with different models can be compared easily.

The glottal waves produced with the new model are analyzed and will be compared with the glottal waves produced by lumped parameter models using the Bernouilli-based model. The comparison concerns the fundamental frequency, glottal peak flow rate, and closed quotient.

To obtain information about the phonation threshold pressure and the range of self-sustained oscillation, the subglottal pressure was increased during the simulations by 8.0 kPa per second until a value of 2 kPa is reached. Simulations have been performed for an initial glottal gap of 0.0, 0.05, 0.01 and 0.25 mm. The initial shape
of the glottis during these simulations was uniform, no diverging and converging initial shapes are simulated.

### 3.3 Results

Using a grid with 128 times 30 cells, the condition of stable solutions is satisfied. Therefore, this grid is used in all the simulations.

As could be expected, using the parameters sets of Ishizaka & Flanagan and De Vries, different results were obtained. Using the first mentioned set, no self-sustained oscillation is obtained for subglottal pressures between 0.0 and 2.0 kPa and different initial glottal gaps. In contrast herewith, the De Vries parameters show self-sustained oscillation for a wide range of subglottal pressure and initial glottal gap. Therefore, the following results are all produced using the De Vries parameters.

The properties of glottal waves for a subglottal pressure of 0.6 kPa have been determined. Glottal waves with a closed phase could only be obtained for an initially closed glottis and a subglottal pressure of 0.6 kPa is applied. Comparison of the properties of the glottal waves with the properties of the glottal waves produced using the Bernoulli-used model instead of the Navier-Stokes equations are summarized in Table II. As a reference, values for normal phonation according to Holmberg et al. (1989) are listed. From this table, it can be seen that the fundamental frequency appears to be lower by using the Navier-Stokes equations than by using the model based on the Bernoulli equation.

<table>
<thead>
<tr>
<th></th>
<th>two-mass + Bernoulli</th>
<th>two-mass + Navier-Stokes</th>
<th>normal phonation</th>
</tr>
</thead>
<tbody>
<tr>
<td></td>
<td>Female</td>
<td>Male</td>
<td></td>
</tr>
<tr>
<td>Fundamental frequency (F_0) (Hz)</td>
<td>187</td>
<td>165</td>
<td>207</td>
</tr>
<tr>
<td>Glottal peak flow (U_g) (l/s)</td>
<td>0.25</td>
<td>0.48</td>
<td>0.14</td>
</tr>
<tr>
<td>Closed quotient (CQ) (-)</td>
<td>0.30</td>
<td>0.30</td>
<td>0.26</td>
</tr>
</tbody>
</table>

The glottal peak flow of the glottal waves produced using the Navier-Stokes equations is a factor two higher than the glottal peak flow of the glottal waves produced using the Bernoulli-based model. This value is also higher than the normal value in female and male phonation as shown in Table II.

Application of Navier-Stokes instead of Bernoulli-based model does not influence the value of the closed quotient.
Results of an increase of subglottal pressure in the model using different values for the initial glottal gap are shown in Fig. 3.

FIG. 3 Glottal waves for different initial glottal openings and rising subglottal pressure; at the bottom right, a close up of one glottal wave (0.1 mm initial gap) is shown.
It can be seen that, using a different initial glottal gap, oscillation starts at a different subglottal pressure. The pressure at which self-sustained oscillation occurs (phonation threshold pressure) is shown in Fig. 4. At this phonation threshold pressure, a sinus-like waveform is obtained. Increasing the pressure above the phonation threshold pressure results in a more than proportional increase of the amplitude of the oscillation, resulting in glottal waves that are not sinus-like but which have a closed quotient. The pressure at which glottal waves with a closed quotient are obtained, are plotted in Fig. 4 also. The phonation threshold pressure obtained is higher using the Navier-Stokes than when using the Bernoulli-based model (Fig. 4) for all initial glottal gaps. Titze (1988) published the phonation threshold pressure for different initial glottal gaps; for a uniform glottis with an initial glottal gap of 0.1 mm, he determined a value of approximately 0.9 kPa, which is close to the value obtained using the Navier-Stokes equations of 1.2 kPa. The value obtained with the Bernoulli-based model is almost 0.1 kPa, which is much lower.

The rate of change of the fundamental frequency as a function of the subglottal pressure ($dF_0/dP_s$) is derived by dividing the difference between $F_0$ at 0.6 kPa ($F_0=165$ Hz) and at 1.6 kPa ($F_0=168$ Hz) by the difference in pressure (1.6-0.6=1.0 kPa). Over this pressure range, $dF_0/dP_s$ is determined to be 3 Hz/kPa, which is almost equal to 0.3 Hz/cm H₂O. This value is substantially lower than the 2.5
Hz/cm H₂O determined by Ishizaka and Flanagan (1972) using their two-mass model. Using the glottal flow model that is based on the Bernoulli equation, the frequency remains unchanged when subglottal pressure is varied.

To demonstrate the variations in glottal flow during a glottal cycle, Fig. 5 shows the velocity component along the main flow direction in an open phase and in a closed phase, as determined during a simulation with the De Vries parameters with an initial glottal opening of 0.5 mm and a subglottal pressure of 0.8 kPa, resulting in a
Chapter 3

mean flow of 430 ml/s. In Fig. 5a it can be seen that the maximum flow velocity in the jet is 40 m/s.

3.4 Discussion

In this study it is demonstrated that self-sustained oscillation can be generated with a two-mass model of the vocal folds in combination with a Navier-Stokes description of the glottal flow. The results show that the choice of the set of parameter values is crucial to achieve phonation: no self-sustained oscillation is obtained using the parameter values of Ishizaka and Flanagan (1972), while the use of the parameter values of De Vries et al. (1999) results in the production of realistic glottal waves. This could be caused by the fact that the parameter values of Ishizaka & Flanagan are larger than those of De Vries. Probably, the values of the masses and springs that represent the vocal folds are over-estimated by Ishizaka & Flanagan, as suggested by Lous et al. (1998). In the case of an over-estimation of the mechanic influence of the vocal folds, the inertial loads and stiffness predominate the dynamic behavior of the vocal folds.

The properties of the glottal waves produced using the Navier-Stokes equations differ from those produced using the Bernoulli-based model. Because applying the Navier-Stokes equations lowers the fundamental frequency, it can be stated that an increase in the effective mass of the vocal folds has been achieved. So the lowering in the fundamental frequency might be partially explained by the influence of inertia effects which are present in the Navier-Stokes equation and absent in the Bernoulli equation. To which extent this effect contributes to the lowering is a question that will be answered in a forthcoming study.

The fact that the glottal peak flow is increased by a factor two by applying the Navier-Stokes equations instead of the Bernoulli-based model can be explained by the fact that the viscous losses are described in a different manner: in the Bernoulli-based model, viscous effects are roughly simulated by the introduction of a fixed separation point for a convergent and divergent glottis. In the Navier-Stokes equations, the viscous effects in the main flow and in the boundary layer are described much more accurately. The closed quotient of the glottal waves produced using the Navier-Stokes equations does not differ from the closed quotient produced using the Bernoulli-based model. The combination of a higher glottal peak flow with an equal closed quotient results in a higher glottal airflow velocity in the Navier-Stokes simulations than from the Bernoulli-based model. This also can be due to a different description of the viscous effects in both aerodynamic models.

The phonation threshold pressure depends on the initial glottal gap. This is in correspondence with Titze (1988), who stated that a tighter adduction of the vocal folds results in a lower phonation threshold pressure. The fact that a sinus-like oscillation occurs at a lower pressure using a gap of 0.05mm can be explained by the fact that the initially closed glottis has to be opened first. The transition of a
sinus-like oscillation to oscillation with a closed phase has also been demonstrated in normal phonation (Schutte and Seidner, 1988).

The value of the maximum jet velocity as shown in Fig. 5 corresponds very well with Alipour (1996a). In an excised larynx, he measured a supraglottal jet velocity of about 40 m/s at almost the same mean flow rate, namely 470 ml/s.

In the model presented in this paper, we are able to apply a subglottal pressure without using the Bernoulli equation to approximate the subglottal pressure for a given flow field. This is in contrast to Alipour and Titze (1996b) and Guo et al. (1993). In comparison with Alipour and Titze (1996b) we can state that, despite of our simple mechanical description of the vocal folds, glottal waves are obtained that are at least as realistic as in their study.

In our study, the vocal folds are sharp-edged because we do not apply any rounding to the geometry of the vocal fold. This choice is made because of the uncertain measures for rounding of the vocal folds that are available. Instead of the recommendations by Alipour and Titze (1999), no bulging of the vocal fold surface is applied. If bulging is applied, the small pulses of air in the closed phase (which occur at higher pressure, as shown in Fig. 3) would probably be avoided. From this point of view, rounding of the vocal fold at the edges and bulging of the vocal fold surface might be a possible improvement for the mechanical model.

3.5 Acknowledgement

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